

AMENDMENTS TO THE SPECIFICATION

Please replace the paragraph beginning on page 2, line 21 with the following rewritten paragraph.

a tray-shaped case having a recessed part which has a sidewall provided with a steam supply port through which steam from the steam generator enters the recessed part, the recessed part having a planar shape that is roughly axisymmetrical-symmetrical with respect to a center line of a steam flow entering through the steam supply port, the tray-shaped case being placed, with an opening of the recessed part directed downward, on the heating chamber at the ceiling steam outlet;

Please replace the paragraph beginning on page 3, line 5 with the following rewritten paragraph.

a heater placed in the recessed part such that a planar shape of the heater is roughly axisymmetrical-symmetrical with respect to said center line; and

Please replace the paragraph beginning on page 3, line 10 with the following rewritten paragraph.

In the steam cooker with the above construction, steam generated by the steam generator is raised in temperature by the steam temperature-raising device to become superheated steam, and the superheated steam is supplied into the heating chamber, whereby the object to be cooked in the heating chamber is heated. At this time, steam generated in the steam generator enters the recessed part of the tray-shaped case of the steam temperature-raising device through the steam supply port provided in the sidewall of the recessed part which is placed above the ceiling steam outlet of the heating chamber. Steam flowing into the recessed part, which has a planar shape (i.e., a shape in a plan view) roughly axisymmetrical-symmetrical with respect to the center line of the steam flow entering through the steam supply port, is divided into the left and right side branches after colliding with a sidewall on the opposite side of the steam supply port of the recessed part, and a part of the branched steam returns along the

sidewalls of the recessed part toward the steam supply port and merges into steam entering through the steam supply port. In this manner, steam filling the recessed part of the tray-shaped case is supplied into the heating chamber via the ceiling steam outlet of the heating chamber located on the opening side of the recessed part.

Please replace the paragraph beginning on page 4, line 9 with the following rewritten paragraph.

The planar shape, i.e., the shape in a plan view, of the recessed part of the tray-shaped case and the planar shape of the heater placed in the recessed part are individually made roughly axisymmetrical-symmetrical with respect to the center line of the steam flow entering by the steam supply port, whereby the steam flow in the recessed part is not biased to the left or right, so that the temperature distribution in the entire recessed part of the tray-shaped case is made uniform.

Please replace the paragraph beginning on page 4, line 18 with the following rewritten paragraph.

Further, the fin pitch of the spiral heat radiation fin wound around the heater, which heater is provided in the recessed part of the tray-shaped case and arranged in a planar shape roughly axisymmetrical-symmetrical with respect to the center line, is set to 10 mm or less, whereby influence (disturbance or resistance loss) of the heat radiation fin on the steam flow jetted from the steam supply port can be distributed in a manner so as to be roughly axisymmetrical-symmetrical with respect to the center line, and the steam temperature distribution in the recessed part is made uniform, while improving the efficiency of heat radiation from the heater by the heat radiation fin. Therefore, by making the temperature distribution of steam supplied into the heating chamber through the ceiling steam outlet of the heating chamber uniform, the temperature distribution in the heating chamber can be made uniform, and uniform cooking can be realized.

Please replace the paragraph beginning on page 5, line 10 with the following rewritten paragraph.

In one embodiment, a plurality of steam outlets for supplying steam into the heating chamber sideways are provided, in roughly ~~axisymmetrical~~symmetrical arrangement with respect to said center line, in sidewalls facing each other across the center line of the recessed part of the tray-shaped case and on the opposite side of the steam supply port.

Please replace the paragraph beginning on page 5, line 17 with the following rewritten paragraph.

In the steam cooker of the above embodiment, the plurality of steam outlets are provided in the sidewalls facing each other across the center line of the recessed part of the tray-shaped case and on the opposite side of the steam supply port, and a part of steam in the recessed part is supplied sideways into the heating chamber via the plurality of the steam outlets. By roughly ~~axisymmetrically~~symmetrically providing the plurality of steam blowout openings in the sidewalls facing each other across the center line of the recessed part of the tray-shaped case and on the opposite side of the steam supply port, the temperature distribution of steam that blows outward toward both sides of the center line via the plurality of steam outlets can be made uniform. Thus, the bias to the left or right in the temperature distribution of steam supplied into the heating chamber sideways can be reduced, so that the temperature distribution in the heating chamber can be made uniform. Furthermore, by providing the plurality of steam outlets in the sidewalls facing each other across the center line of the recessed part of the tray-shaped case and on the opposite side of the steam supply ports, steam of the highest flow rate and temperature (due to heating by the heater) in the recessed part blows out from the plurality of steam outlets. This vigorously supplies high-temperature steam into the heating chamber. Thereby, steam in the heating chamber is agitated, which is preferred in resolving uneven cooking.

Please replace the paragraph beginning on page 32, line 21 with the following rewritten paragraph.

A first and a second non-heating portion 52b, 52c of the first steam superheater 52 pass through the sidewall 91 and fixed at the outside of the steam supply pipes 94A, 94C. Electric wirings (not shown) are connected to leading ends of the first and the second non-heating portion 52b, 52c of the first steam superheater 52. The first steam superheater 52 has a planar shape roughly axisymmetrical-symmetrical with respect to the center line L of the flow of steam flowing in from the steam supply ports 95A, 95B, 95C, and includes the two non-heating portions 52b, 52c placed parallel to the center line L at a predetermined interval, two roughly U-shaped heating portions 52a-1, 52a-2 which are connected, at one end thereof, to the leading ends of the non-heating portions 52b, 52c and which are each curved toward the center of the recessed part 51a, and a roughly U-shaped third heating portion 52a-3 which connects the two first and second heating portions 52a-1, 52a-2. Spiral heat radiation fins 56 are provided around the first to third heating portions 52a-1 to 52a-3 and a part of the first and second non-heating portions 52b, 52c of the first steam superheater 52.

Please replace the paragraph beginning on page 33, line 17 with the following rewritten paragraph.

Non-heating portions 53b, 53c at both ends of the second steam superheater 53 pass through the sidewall 91 and are fixed between the steam supply pipes 94A and 94B and between the steam supply pipes 94B and 94C, respectively. Electric wirings (not shown) are connected to leading ends of the non-heating portions 53b, 53c of the second steam superheater 53. A heating portion 53a of the second steam superheater 53 has a circular shape and both ends of the circular heating portion 53 have a shape continuous with the non-heating portions 53b, 53c. The second steam superheater 53 has a planar shape roughly axisymmetrical-symmetrical with respect to the center line L of the flow of steam flowing in from the steam supply ports 95A, 95B, and 95C.

Please replace the paragraph beginning on page 37, line 12 with the following rewritten paragraph.

Fig. 11(a) shows a view for describing heat radiation fins of the steam temperature-raising device of the steam cooker of the invention. This shows parts of the first and second heating portions 52a-1, 52a-2, which are parallel to each other, of the first steam superheater 52 in the vicinity of the first sidewall 91 of the tray-shaped case 51 (shown in Figs. 7(a), (b)). Since the spiral heat radiation fins 56 are provided in the same winding direction around the first steam superheater 52, inclined planes of the heat radiation fins 56 are inclined in the same direction. Therefore, for example, influence of the inclined planes of the heat radiation fins 56 on steam flowing from the upper side to the lower side in the figure through the center of the first and second heating portions 52a-1, 52a-2, which are parallel to each other, of the first steam superheater 52 shown in Fig. 11(a) varies. However, in this embodiment, the fin pitch of the spiral heat radiation fins 56 that are provided around the first steam superheater 52 is set to 10 mm, whereby the distribution of the degree of influence (disturbance or resistance loss) of the heat radiation fins 56 on the steam flow blowing out from the steam supply ports 94A, 94B, 94C (shown in Fig. 9) is made axisymmetrical-symmetrical with respect to the center line L of steam.

Please replace the paragraph beginning on page 39, line 7 with the following rewritten paragraph.

In this manner, according to the steam cooker with the above construction, the planar shapes of the recessed part 51a of the tray-shaped case 51 and the first steam superheater 52 placed in the recessed part 51a thereof are made roughly axisymmetrical-symmetrical with respect to the center line L of the flow of steam flowing in from the steam supply ports 95A, 95B, 95C, whereby steam flow in the recessed part 51a is not biased or deviated, the temperature distribution thereof is made uniform, and the temperature distribution of steam supplied into the heating chamber 20 from the ceiling steam outlets 55 of the ceiling panel 54 of the heating chamber 20 can be made uniform.

Please replace the paragraph beginning on page 39, line 19 with the following rewritten paragraph.

Further, the fin pitch of the spiral heat radiation fins 56, which are wound around the first steam superheater 52 placed in the recessed part 51a of the tray-shaped case 51, is set to 10 mm or less, whereby the degree of influence (disturbance or resistance loss) on the steam flow blowing out from the steam supply ports 95A, 95B, 95C can be made roughly axisymmetrical symmetrical with respect to the center line L. Consequently, the temperature distribution of steam in the recessed part 51a is made more uniform, and the temperature distribution of steam supplied into the heating chamber 20 from the ceiling steam outlets 55 of the ceiling panel 54 of the heating chamber 20 can be made uniform. Therefore, the temperature distribution in the heating chamber 20 can be made uniform, which makes it possible to perform uniform cooking.

Please replace the paragraph beginning on page 40, line 9 with the following rewritten paragraph.

By providing the steam outlets 101A-104A, 101B-104B in the second and the third sidewalls of the recessed part 51a, respectively, which face each other across the center line L, and on the opposite side of the steam supply ports 95A, 95B, 95C, and arranging the steam outlets 101A-104A, 101B-104B in the recessed part 51a so that they are roughly axisymmetrical symmetrical with respect to the center line L, the amount, flow rate and temperature distribution of steam blowing sideways to the left or right from the inside of the recessed part 51a via the steam outlets 101A-104A, 101B-104B, can be made uniform. Thus, bias or unevenness in the temperature distribution of steam supplied sideways into the heating chamber 20 via the steam supply passage 23 (shown in Fig. 3) can be reduced, so that the temperature distribution in the heating chamber 20 can be made more uniform.

Please replace the paragraph beginning on page 41, line 1 with the following rewritten paragraph.

Even if the planar shape of the recessed part of the tray-shaped case and the planar shape of the heater 50 are not-axisymmetrical symmetrical, by providing the plurality of

steam outlets for supplying steam sideways into the heating chamber in both of the sidewalls adjacent to the sidewall provided with the steam supply ports of the recessed part of the tray-shaped case on the opposite side of the steam supply ports, the temperature distribution of steam blowing outward to both sides via the plurality of steam outlets can be made uniform.

Please replace the paragraph beginning on page 41, line 23 with the following rewritten paragraph.

In the roughly pentagonal recessed part 51a of the tray-shaped case 51, the two sides that are continuous with both ends of the U-shape of the three sides and are directed outward to form a V-shape serve as the fourth and fifth sidewalls 93A, 93B, which are inclined with respect to the plane roughly perpendicular to the center line L. Thereby, when steam flowing into the recessed part 51a of the tray-shaped case 51, whose planar shape is axisymmetrical-symmetrical with respect to the center line L, collides with the fourth and fifth sidewalls 93A, 94B and is divided into the left and right side branches, the steam branch flows are made smooth, generation of steam accumulation and the like, which causes uneven temperature distribution, is suppressed, and the temperature distribution of steam flowing in the recessed part 51a can be made more uniform.

Please replace the paragraph beginning on page 46, line 21 with the following rewritten paragraph.

Of the first and second superheaters 52, 53 with different power densities per unit surface area and having a planar shape that is axisymmetrical-symmetrical with respect to the center line L of the steam flow entering from the steam supply ports 95A, 95B, 95C, the second steam superheater 53 having a high power density per unit surface area is placed inside, while the first steam superheater 52 having a low power density per unit surface area is placed outside, whereby low-temperature steam flowing in from the steam supply ports 95A, 95B, 95C is heated by the second steam superheater 53 having a high power density per unit surface area, which is placed inside, and, in its outside, steam is heated by the first steam superheater 52 having

a low power density per unit surface area. Therefore, the temperature distribution in the entire recessed part 51a of the tray-shaped case 51 can be made uniform more effectively.

Please replace the paragraph beginning on page 47, line 12 with the following rewritten paragraph.

Due to the use of the first and second steam superheaters 52, 53 having a planar shape that is axisymmetrical-symmetrical with respect to the center line L of the steam flow entering from the steam supply ports 95A, 95B, 95C, steam flow is not biased toward the left or right side so that the temperature distribution in the recessed part 51a of the tray-shaped case 51 can be made uniform effectively.

Please replace the paragraph beginning on page 47, line 20 with the following rewritten paragraph.

The first steam superheater 52 has the two first and second non-heating portions 52b, 52c that are spaced from each other and placed roughly parallel to the center line L, the two roughly U-shaped first and second heating portions 52a-1, 52a-2 which are connected, at one end thereof, to leading ends of the first and second non-heating portions 52b, 52c, respectively, and which are each curved toward the center of the recessed part 51a into a U shape, and the roughly U-shaped third heating portion 52a-3 that connects between the other ends of the first and second heating portions 52a-1, 52a-2, and the first steam superheater 52 has a planar shape that is axisymmetrical-symmetrical with respect to the center line of the steam flow entering from the steam supply ports 95A, 95B, 95C. Thus, steam flow in the recessed part 51a is not biased, so that the temperature distribution in the entire recessed part 51a of the tray-shaped case 51 can be made uniform effectively.

Please replace the paragraph beginning on page 48, line 21 with the following rewritten paragraph.

In the above embodiment, the first steam superheater 52 (shown in Fig. 7) as a planar heater is used for the steam temperature-raising device 50. The shape of the heater is not

limited to the above-described one, and any heater may be used as long as it has a planar shape that is roughly axisymmetrical-symmetrical with respect to the center line L of the steam flow flowing in from the steam supply ports.

Please replace the paragraph beginning on page 49, line 10 with the following rewritten paragraph.

One end (on the side of the non-heating portion) of the linear portion 58a is inserted through the first side wall 91 of the tray-shaped case 51 in the vicinity of one end of the first sidewall 91, and the other end of the linear portion 58a extends approximately parallel to the second sidewall 92A up to the vicinity of the fourth sidewall 93A. The other end of the linear portion 58a is then connected to one end of the semi-circular curved portion 58b, and the other end of the semi-circular curved portion 58b is connected to one end of the linear portion 58c, which extends approximately parallel to the linear portion 58a to the side of the first sidewall 91. The other end of the linear portion 58c is connected to one end of the semi-circular curved portion 58d, and the other end of the semi-circular curved portion 58d is connected to one end of the linear portion 58e, which extends approximately parallel to the linear portion 58c to the opposite side of the first sidewall 91. The other end of the linear portion 58e is connected to one end of the semi-circular curved portion 58f, and the other end of the linear portion 58f is connected to one end of the linear portion 58g, which extends approximately parallel to the third sidewall 92B. The other end (on the side of the non-heating portion) of the linear portion 58g is inserted through the first sidewall 91 in the vicinity of the other end of the first sidewall 91. The heater 58 has a planar shape that is roughly axisymmetrical-symmetrical with respect to the center line L of the steam flow entering from the steam supply ports 95A, 95B, 95C.